Implementing the Relaxation Damping of Air into the Navier-Stokes Equation

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Nomenclature

c = velocity of sound, m/s d = characteristic length, m f = frequency, Hz

 $f_{\rm O}$, $f_{\rm N}$ = relaxation frequencies of oxygen and nitrogen, Hz

h = molar concentration of water vapor, %

 $G_{O,N}$ = damping constants of oxygen and nitrogen, s/m

 $I = 3 \times 3$ identity tensor

p = pressure, pressure of sound, N/m² S = stress tensor, N/m²

S = stress tensor, N/m² St = Strouhal number T = temperature, K t = time, s

 t^* = integration time v = flow velocity in air, m/s $W_{\rm dil}$ = dilatations tensor, 1/s $W_{\rm dist}$ = distortion tensor, 1/s

 x, x_i = Cartesian coordinates, m, i = 1, 2, 3

 α = sound attenuation, 1/m η = shear viscosity, kg/ms

 $\begin{array}{ll} \lambda & = \mbox{vibrational relaxation viscosity, kg/ms} \\ \mu & = \mbox{rotational relaxation viscosity, kg/ms} \end{array}$

 ρ = density of air, kg/m³ τ = tension, N/m²

Subscripts

N = nitrogen O = oxygen R = rotation

0 = standard atmosphere

I. Introduction

R EDUCING the Navier-Stokes equation to small flow velocities, the acoustic wave equation can be obtained. This equation contains an acoustic absorption which is proportional to the shear viscosity. Kirchhoff added a second absorption effect, due to the irreversible heat dissipation within the acoustic wave. These two terms describe the classical Stokes-Kirchhoff absorption in gases. But in 1933, when Knudsen¹ carried out experiments in air under atmospheric conditions, he discovered a much higher level of sound absorption. This absorption exceeded by a factor of about 20 the absorption predicted by Stokes and Kirchhoff. In the same year Kneser² explained the discrepancy by introducing a third damping phenomenon, that of relaxation. In particular, Kneser found that the vibrational relaxation of oxygen is responsible for enhanced acoustic attenuation in air and in the infrasonic range, the vibrational relaxation of nitrogen. Moreover, by using this theory it was possible to explain the sensitive influence of natural humidity on acoustic absorption in the air. Although the vapor is a small fraction in air it changes the acoustic absorption by orders of magnitude. The water vapor catalyzes the stimulation of the vibrational degrees of freedom. But not only the vibration, also the rotation of air molecules cause relaxation effects.

The task of this investigation is to implement the theoretically and experimentally proven relaxation damping into the Navier-Stokes equation for air.

II. Sound Absorption

To obtain the acoustic absorption due to the shear viscosity η , the classical Navier-Stokes equation

$$\rho\left(\frac{\partial v}{\partial t} + v \cdot \nabla v\right) = -\operatorname{grad} p + \eta \Delta v + \frac{1}{3}\eta \operatorname{grad} \operatorname{div} v \tag{1}$$

is linearized. Using the linearized material and continuity equations, the acoustic equation including viscous shear effects is obtained

$$\frac{\partial^2 p}{\partial t^2} - c^2 \Delta p - \frac{4\eta}{3\rho} \frac{\Delta \partial p}{\partial t} = 0$$
 (2)

A plane sound wave propagating in the direction x with frequency f

$$p = p_0 \exp(-\alpha_s x) \sin 2\pi f(t - x/c)$$
 (3)

substituted in the equation for sound pressure, Eq. (2) yields the relation between the shear viscosity η and the attenuation of sound $\alpha_s(\alpha_s \ll 2\pi f/c)$

$$\alpha_s = 8\pi^2 f^2 \eta / 3\rho c^3 \tag{4}$$

The shear mechanism causes only a small portion of the atmospheric sound absorption. There is also the Kirchhoff attenuation by heat dissipation, but the relaxation effects are dominant. First, the rotational relaxation of the air molecules gives the sound absorption^{4,5}

$$\alpha_R = 2\pi^2 f^2 \mu / \rho c^3 \tag{5}$$

with μ as the rotational relaxation viscosity. According to the noise measurements by Greenspan³ this viscosity in air is $\mu=O,6\eta$. The main attenuation in the atmosphere up to aerodynamically relevant frequencies is caused by relaxation of the vibrational degrees of freedom. This relaxation process is characterized by the damping constants G_O and G_N and the relaxation frequencies f_O and f_N and gives rise to the acoustic attenuations α_N and α_O for the atmospheric nitrogen and oxygen^4,5

$$\alpha_{\rm O} = G_{\rm O} f^2 / (f_{\rm O} + f^2 / f_{\rm O})$$

$$\alpha_{\rm N} = G_{\rm N} f^2 / (f_{\rm N} + f^2 / f_{\rm N})$$
(6)

Numerous measurements on acoustic attenuation in air have been made. In the following, the International Standard Organization-norm⁶ draft is applied. According to Ref. 6 the numeric values for the damping constants $G_{\rm O}$ and $G_{\rm N}$ and the relaxation frequencies $f_{\rm O}$ and $f_{\rm N}$ are

$$G_{\rm O} = 0.01275 (T/T_0)^{-5/2} \exp(-2239.1/T)$$

$$G_{\rm N} = 0.1068 (T/T_0)^{-5/2} \exp(-3352.0/T)$$
(7)

$$f_{\rm O} = (p/p_0)\{24 + (4.04 \cdot 10^4)(h)[(0.02 + h)/(0.391 + h)]\}$$

$$f_{\rm N} = (p/p_0)(T/T_0)^{-1/2}[9 + 280h \exp\{-4.170[(T/T_0)^{-1/3} - 1]\}]$$
(8)

where p_0 and T_0 are the reference values of the standard atmo-

$$p_0 = 101.325 \text{ kPa} (= 1 \text{ atm}), \quad T_0 = 293.15 \text{ K} (= 20^{\circ}\text{C}) \quad (9)$$

The effects of temperature T and molar concentration of water vapor h on the relaxation frequencies $f_{\rm O}$ and $f_{\rm N}$ are presented in Table 1. The surprising effect is the strong catalytic influence of air humidity. A molar concentration of h=1% (corresponding to a relative humidity of 43% at 20°C) increases the relaxation frequency $f_{\rm O}$ of oxygen from 24 to 29,600 Hz.

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Table 1 Vibrational relaxation frequencies f_0 and f_N and vibrational viscosities λ_0 and λ_N for oxygen and nitrogen in standard atmosphere

Standard atmosphere ISO standard ⁶				Vibr. relaxation frequency		Vibr. relaxation viscosity	
H, km	<i>T</i> , K	P, kPa	h, %	$f_{\rm O}, {\rm Hz}$	f _N , Hz	$\lambda_{\rm O}$, kg/ms	λ _N , kg/ms
0	288	101.3	1.009	29,978	287.09	4.593×10^{-4}	8.428×10^{-3}
2	275	79.5	0.606	12,079	132.94	6.793×10^{-4}	9.036×10^{-3}
4	262	61.6	0.358	4,452	60.77	1.051×10^{-3}	9.217×10^{-3}
6	249	47.2	0.098	456.2	15.53	5.566×10^{-3}	1.568×10^{-2}
8	236	35.6	0.016	28.52	4.808	4.566×10^{-2}	2.027×10^{-2}
10	223	26.4	0.006	10.39	3.026	5.974×10^{-2}	1.169×10^{-2}

^aAccuracy better than 0.1%. Rotational relaxation viscosity in air is $\mu = 0.6\eta$ (Ref. 4, p. 553).

III. Viscous Stress Tensors

The differential motion of a fluid with the velocity $v = v(x_i, t)$ is given by the velocity tensor $\nabla v = \{\partial v_i/\partial x_j\}$. The antisymmetric part of ∇v describes the flow rotation and needs not to be considered here. The symmetric tensor ∇v_{sym} consists of the distortion tensor W_{dist} and the dilatation tensor W_{dil} . This corresponds mathematically to the separation of ∇v_{sym} in a tensor W_{dist} with zero spur and in a spherically symmetric tensor W_{dil} (∇v^T = transponded tensor, $I = 3 \times 3$ identity tensor)

$$(\nabla v)_{\text{sym}} = (\nabla v + \nabla v^T)/2 = W_{\text{dist}} + W_{\text{dil}}$$
 (10)

$$W_{\text{dist}} = (\nabla v)_{\text{sym}} - (1/3)I \text{ div } v \tag{11}$$

$$W_{\text{dil}} = (1/3) I \text{ div } v \tag{12}$$

According to the Stokes hypothesis that the viscous stresses τ_{ij} are solely caused by the distortion tensor W_{dist} , representing the constant volume behavior, the stress tensor S_s is defined by (the factor 2 was introduced by Stokes to obtain Newton's shear stress law)

$$S_s = 2\eta W_{\text{dist}} \tag{13}$$

The total volume change in the fluid is given by the dilatation tensor $W_{\rm dil}$. As the relaxation damping is proportional to the change of temperature, and therefore to the volume change, the stress tensor S_R describing rotational relaxation gives

$$S_R = 3\mu W_{dil} = \mu I \text{ div } v \tag{14}$$

With the same argumentation it is postulated that the vibrational relaxation stresses are also proportional to $W_{\rm dil}$. The formal proportional factors λ_0 and $\lambda_{\rm N}$ and can be identified later as the vibrational relaxation viscosities of the oxygen and nitrogen. Additionally, the frequency dependance (6) of vibration relaxation is to be considered. The transformation from the frequency domain into the time domain is done by convolution. Especially, the frequency behavior according to Eq. (6) corresponds to

$$f^2/(f^2 + f_{O,N}^2) \to \exp(-2\pi f_{O,N} t^*)$$
 (15)

and gives the stress tensors $S_{\rm O}$ and $S_{\rm N}$ of the vibrational relaxation for the atmospheric oxygen and nitrogen

$$S_{\rm O} = \lambda_{\rm O} I \int_{t^*=0}^{\infty} \exp(-2\pi f_{\rm O} t^*) \operatorname{div} v(t-t^*) \, d2\pi f_{\rm O} t^*$$

$$(16)$$

$$S_{\rm N} = \lambda_{\rm N} I \int_{t^*=0}^{\infty} \exp(-2\pi f_{\rm N} t^*) \operatorname{div} v(t-t^*) \, d2\pi f_{\rm N} t^*$$

The variable for integration t^* has the dimensions of time and represents the retarded time, starting at the present time t. Physically, the relaxation tensor depends on the present velocity v(t), as well as the past $v(t-t^*)$.

IV. Flow Equation for Air

The flow equation for air with the viscous stress tensors S_S , S_R , S_O , and S_N is given by

$$\frac{\rho \partial v}{\partial t} + \rho v \cdot \nabla v = -\operatorname{grad} p + \operatorname{div} S_S + \operatorname{div} S_R + \operatorname{div} S_O + \operatorname{div} S_N$$
(17)

Assuming constant shear and relaxation viscosities η , μ , λ_0 , and λ_N , the formal calculation gives the flow equation for the atmospheric air

$$\frac{\rho \partial \nu}{\partial t} + \rho \nu \cdot \nabla \nu = -\operatorname{grad} p$$

$$+ \eta \left(\Delta \nu + \frac{1}{3} \operatorname{grad} \operatorname{div} \nu \right) + \mu \operatorname{grad} \operatorname{div} \nu$$

$$+ \lambda_0 \int_{t^*=0}^{\infty} \exp(-2\pi f_0 t^*) \operatorname{grad} \operatorname{div} \nu (t - t^*) d2\pi f_0 t^*$$

$$+ \lambda_N \int_{t^*=0}^{\infty} \exp(-2\pi f_N t^*) \operatorname{grad} \operatorname{div} \nu (t - t^*) d2\pi f_N t^* \qquad (18)$$

The first line corresponds to the Navier-Stokes equation; the additional terms reflect viscous relaxation forces for atmospheric air. The convolution integrals appear complicated, but they can be evaluated numerically. The calculation process gives knowledge about the past $v(t-t^*)$, which needs not be calculated additionally. Also the integral converges very fast due to the exponential convolution function.

To determine the unknown relaxation viscosities λ_0 and λ_N , and also to validate, the flow equation (18) will be transformed by linearization, and using the material and the continuity conditions in the equation for the sound pressure ($\dot{p} = \partial p/\partial t$)

$$\ddot{p}/c^{2} = \Delta p + (4\eta/3\rho c^{2} + \mu/\rho c^{2}) \Delta \dot{p}$$

$$+ \left(\frac{\lambda_{O}}{\rho c^{2}}\right) \int_{t^{*}=0}^{\infty} \exp(-2\pi f_{O} t^{*}) \Delta \dot{p} (t - t^{*}) d2\pi f_{O} t^{*}$$

$$+ \left(\frac{\lambda_{N}}{\rho c^{2}}\right) \int_{t^{*}=0}^{\infty} \exp(-2\pi f_{N} t^{*}) \Delta \dot{p} (t - t^{*}) d2\pi f_{N} t^{*}$$
(19)

A plane acoustic wave in the x direction with the frequency f

$$p = p_0 \exp(-\alpha_S x - \alpha_R x - \alpha_O x - \alpha_N x) \sin 2\pi f (t - x/c)$$
 (20)

inserted in Eq. (19) one gets the sound attenuations α_S by shear, α_R by rotational, and α_O and α_N by vibrational relaxation

$$\alpha_S = \frac{8\pi^2 \eta f^2}{3\rho c^3}, \qquad \alpha_R = \frac{2\pi^2 \mu f^2}{\rho c^3}$$
 (21)

$$\alpha_{\rm O} = \frac{2\pi^2 \lambda_{\rm O} f_{\rm O} f^2}{\rho c^3 (f_{\rm O} + f^2 / f_{\rm O})}, \qquad \alpha_{\rm N} = \frac{2\pi^2 \lambda_{\rm N} f_{\rm N} f^2}{\rho c^3 (f_{\rm N} + f^2 / f_{\rm N})}$$
 (22)

Comparing Eq. (22) with Eq. (6) one gets the relaxation viscosities of oxygen and nitrogen λ_O and λ_N in the air.

$$\lambda_{\rm O} = \frac{\rho c^3 G_{\rm O}}{2\pi^2 f_{\rm O}}, \qquad \lambda_{\rm N} = \frac{\rho c^3 G_{\rm N}}{2\pi^2 f_{\rm N}}$$
 (23)

In Table 1 the numerical values λ_O and λ_N of the standard atmosphere calculated with Eqs. (7), (8), and (23) are given.

V. Approximated Flow Equation in Air

As an alternative to the exact solution of Eq. (18), the convolution integrals may be substituted by Taylor series expansions with

$$v(t-t^*) = \sum_{n=0}^{\infty} \frac{1}{n!} \frac{\mathrm{d}^n v}{\mathrm{d}t^n} (-t^*)^n = v - \left(\frac{\partial v}{\partial t} + v \nabla v\right) t^* \pm \dots$$
(24)

The flow equation in air is as follows:

$$\frac{\rho \partial \nu}{\partial t} + \rho \nu \cdot \nabla \nu = -\operatorname{grad} p + \eta \left(\nabla \nu + \frac{1}{3} \operatorname{grad div} \nu \right)
+ \mu \operatorname{grad div} \nu + \lambda_{O} \operatorname{grad div} \sum_{n=0}^{\infty} \left(\frac{-1}{2\pi f_{O}} \right)^{n} \frac{\mathrm{d}^{n} \nu}{\mathrm{d} t^{n}}
+ \lambda_{N} \operatorname{grad div} \sum_{n=0}^{\infty} \left(\frac{-1}{2\pi f_{N}} \right)^{n} \frac{\mathrm{d}^{n} \nu}{\mathrm{d} t^{n}}$$
(25)

For air flow under environmental conditions, the zeroth-order term n=0 may be sufficient. For normal air humidity values, the relaxation frequencies are well separated; for nitrogen, $f_{\rm N}=100-300$ Hz, whereas for oxygen, $f_{\rm O}=20,000-30,000$ Hz. Defining the Strouhal numbers $St_{\rm N}$ and $St_{\rm O}$ with d_c the characteristic length and v_c the characteristic velocity

$$St_{N} = f_{N} d_{c} / v_{c}, \qquad St_{O} = f_{O} d_{c} / v_{c}$$
 (26)

For flow velocities with a Strouhal number $St = fd_c/v_c$ the flow equation (25) can be approximated by

$$\frac{\rho \partial \nu}{\partial t} + \rho \nu \cdot \nabla \nu = -\operatorname{grad} p + \eta \Delta \nu$$

$$+ \left\{ \begin{array}{l} (1/3)\eta + \mu + \lambda_{O} + \lambda_{N} \\ (1/3)\eta + \mu + \lambda_{O} \\ (1/3)\eta + \mu \end{array} \right\} \operatorname{grad} \operatorname{div} \nu$$

$$\operatorname{for} \quad \left\{ \begin{array}{l} St < St_{N} \\ St_{N} < St < St_{O} \\ St > St_{O} \end{array} \right\} \tag{27}$$

In the standard atmosphere at sea level the shear viscosity is $\eta = 1.7 \times 10^{-5}$ kg/ms, and the relaxation viscosity of atmospheric oxygen $\lambda_{\rm O} = 4.6 \times 10^{-4}$ kg/ms. In this case the term grad div ν is 79.2 times higher than in the classical Navier-Stokes equation without the vibrational relaxation effect.

References

¹Knudsen, V. O., "The Absorption of Sound in Air, in Oxygen, and in Nitrogen—Effects of Humidity and Termperature," *Journal of the Acoustical Society of America*, Vol. 5, Oct. 1933, pp. 112–121.

²Kneser, V. O., "The Interpretation of the Anomalous Sound-Absorption in Air and Oxygen in Terms of Molecular Collisions," *Journal of the Acoustical Society of America*, Vol. 5, Oct. 1933, pp. 122–132.

³Greenspan, M., "Rotational Relaxation in Nitrogen, Oxygen, and Air," Journal of the Acoustical Society of America, Vol. 31, 1959, pp. 155–160. ⁴Pierce, A. D., Acoustics, McGraw-Hill, New York, 1989, pp. 508–565.

⁵Thompson, P. A, Compressible-Fluid Dynamics, McGraw-Hill, New York, 1972.

⁶Anon., "Acoustics-Attenuation of Sound During Propagation Outdoors," Pts. 1 & 2, International Standard Organization, ISO/TC 43/SC 1/WG 24, Central Secretariat, CH-1211 Geneva 20, Switzerland, Aug. 1988.

Time-Of-Flight Mass Spectrometer for Impulse Facilities

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Introduction

T present, impulse facilities such as shock tunnels and expan-A sion tubes are the only practical ground-based means of simulating aerodynamic conditions at speeds in excess of 3.5 km/s. The duration of test flow, however, is of the order of a millisecond, requiring specialized instrumentation. It is especially desirable to measure species concentrations in the test flow since the chemical state of the test gas may be altered by the high transient temperatures produced. Also, the flow of test gas in a reflected shock tunnel is followed by a flow of driver gas, usually of a different composition. The breakdown of the interface between these two gas slugs causing mixing of the driver and test gases typically limits the available test time. Measurements of species concentrations are the most sensitive methods of determining the duration of test flow and have been used in the past. Furthermore, studies of supersonic combustion ramjets would benefit from measurements of species concentrations downstream of fuel injection.

A quadrupole mass spectrometer was used by Crane and Stalker² in a reflected shock tunnel. They used a two-stage hollow conical skimmer gas sampling system. The short duration of the flow meant that the ions from only a single species of molecule could be recorded during each test flow. To obtain relative concentrations, Crane and Stalker relied upon sampling multiple test flows and measured the total ion production of each shot for normalization. This method assumed repeatability of the test flow and of the behavior of the mass spectrometer despite variations in total ion production. They estimated errors in their relative species concentration measurements of 30%.

Experimental Design

The instrument reported here was designed for use in T4, an existing free piston reflected shock tunnel.³ At the upstream end are three coaxial, hollow conical skimmers used to sample the flow (Fig. 1). Behind the second and third skimmers are vacuum chambers evacuated by diffusion pumps. There is no vacuum chamber behind the first skimmer as the flow through it vents back out into the freestream flow. Gas passing through the skimmers is expanded and collimated into a molecular beam entering the interior high vacuum chamber. At 20 mm behind the final skimmer, an electron beam intersects the molecular beam, causing ionization of some molecules. The ions produced are accelerated along a meter-long flight tube to an electron multiplier particle detector. The signal from this particle detector is recorded on a digital storage oscilloscope.

An essential aspect of the success of the instrument is that the production of the molecular beam be achieved as quickly as possi-

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